

#### 부록 (APPENDIX)

$$T = T_s \times N = \frac{N}{F_s} = \frac{N}{2.56 \times F_{max}} = \frac{lines}{F_{max}}$$

T = Time required to collect the waveform (시간파형을 수집하는데 요구되는 시간)

Ts = Time between each sample (각각의 샘플 사이 시간)

Fs = Sampling rate = Samples per second (샘플링 주파수, 초당 샘플 수)

N = Number of samples (1024, 2048, 4096, etc.) (샘플 수(개수))

# $Resolution = \frac{F_{max}}{lines}$

#### **Bandwidth** = **Resoution** × **Window** factor

Resolution (분해능), Bandwidth (대역폭), Fmax (최대 주파수), lines (라인수)

Window factor (창 인자)= 1 (no window/uniform/rectangular) 또는 1.5 (Hanning window)

Separating Frequency (분리 가능한 주파수) ≥ 2 x Bandwidth (대역폭) ≥ 2 x Resolution (분해능) \* Window Factor (창 인자)

요구되는 스펙트럼 라인 ≥ 2 x Window factor x Fmax / Separating Frequency Accuracy of Frequency (at peak) (피크에서 주파수의 확실성) = ± ½ x Resolution

Prime numbers (소수): 1, 3, 5, 7, 11, 13, 17, 19…

1 inch = 25.4 mm 1mm = 0.039 inches

#### Trial weight calculation (시험 추 계산):

$$W = \frac{F}{K \times R \times N^2}$$

F = 10% of rotor mass divided by the number of bearings (회전자 질량의 10%를 베어링 숫자로 나눔) in kg (단위: kg) K = 0.011 N = RPM/1000 R = Radius (반경) in cm (단위: cm)

#### Unit Conversions (단위 변환)

$D_{pk-pk} = \frac{19098  V_{pk}}{f_{cpm}} \qquad V$	$V_{pk} = \frac{5217  A_{rms}}{f_{cpm}}$	$D_{pk-pk} = \frac{27009  V_{rms}}{f_{cpm}}$	$V_{rms} = \frac{93712  A_{rms}}{f_{cpm}}$		
$D_{pk-pk} = \frac{9.958 \times 10^7 A_{rms}}{f_{cpm}^2} \ A$	$A_{rms} = \frac{f_{cpm}V_{pk}}{5217}$	$D_{pk-pk} = \frac{2.53 \times 10^9 A_{rms}}{f_{cpm}^2}$	$A_{rms} = \frac{f_{cpm}V_{rms}}{93712}$		
$V_{pk} = \frac{f_{cpm}D_{pk-pk}}{19098} \qquad A$	$A_{rms} = \frac{f_{cpm}^2 D_{pk}}{9.958 \times 10^7}$	$V_{rms} = \frac{f_{cpm}D_{pk-pk}}{27009}$	$A_{rms} = \frac{f_{cpm}^2 D_{pk-pk}}{2.53 \times 10^9}$		
<b>D = Displacement:</b> mils pk-pk		D = Displacement: micron pk-pk			
V = Velocity: in/sec pk		V = Velocity: mm/sec rms			
A = Acceleration: g rms		A = Acceleration: g rms			
F = 주파수: CPM		F = 주파수: CPM			
		1 g rms = 9.8m/sec <sup>2</sup>			

### ISO 10816-3 Vibration Severity Chart



If the lowest natural frequency of the combined machine and support system in the direction of measurement is higher than its main excitation frequency (this is in most cases the rotational frequency) by at least 25 %, then the support system may be considered rigid in that direction. All other support systems may be considered flexible.

## ISO 10816-3 Vibration Severity Chart

	6			- 140	5.51	Di	
	D			- 113	4.45	qs	
				- 90	3.54	lac	
				- 71	2.80	ë	
				- 57	2.24	ne	
	В			- 45	1.77	Int	
				- 37	1.46	2 10	
				- 29	1.14	)-100 -100	
				- 22	0.87	i0Hz	
	Α			- 18	0.71	r>6	
				- 11	0.43	00rp 20rp	
				µm rms	mil rms	33	
rigid	flexible	rigid	flexible	Foundation			
medium sized machines 15kW < P ≤ 300kW		large machines 300kW < P < 50MW		Machine Type			
motors 160mm ≤ H < 315mm		motors 315mm ≤ H					
Group 2		Group 1		Group			
A New machine condition Short-term operation allowable							
B Unlimited long-term operation allowable D Vibration causes damage							

Transducer Operating Regions (트랜스듀스 운전 범위)

Frequency



**Frequency Range** 



Transducer Frequency Response Curves (트랜스듀스의 주파수 응답 곡선)



## Transducer Effectiveness Regions (트랜스듀스의 효율적인 사용 범위)



3 Electro-mechanical velocity transducer



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Stage One: Airborne Ultrasound, Shock Pulse, PeakVue, Spike Energy, Envelope



Stage Three: Outer race fault (inner race rotating)



Stage Three: Inner race fault (inner race rotating)



Stage Four







Stage Two: Envelope, Demodulation and Acceleration Spectrum



Stage Three: Outer race fault (outer race rotating)



Stage Three: Ball or roller fault (inner race rotating)