

CERTIFICATION EXAM REFERENCE MATERIAL

$$T = T_s \times N = \frac{N}{F_s} = \frac{N}{2.56 \times F_{max}} = \frac{\text{lines}}{F_{max}}$$

T = Time required to collect the waveform

T_s = Time between each sample

F_s = Sampling rate = Samples per second

N = Number of samples (1024, 2048, 4096, etc.)

$$\text{Resolution} = \frac{F_{max}}{\text{lines}}$$

$$\text{Bandwidth} = \text{Resolution} \times \text{Window factor}$$

Window factor = 1 (no window/uniform/rectangular) or 1.5 (Hanning window)

Separating frequency ≥ 2 x Bandwidth ≥ 2 x Resolution * Window Factor

Required spectral lines ≥ 2 x Window factor x F_{max} / Separating frequency

Accuracy of frequency (at peak) = ± 1/2 x Resolution

Prime numbers: 1, 2, 3, 5, 7, 11, 13, 17, 19...

1 inch = 25.4 mm

1mm = 0.039 inches

Trial weight calculation:

$$W = \frac{F}{K \times R \times N^2}$$

F = 10% of rotor mass divided by the number of bearings in kg

K = 0.011

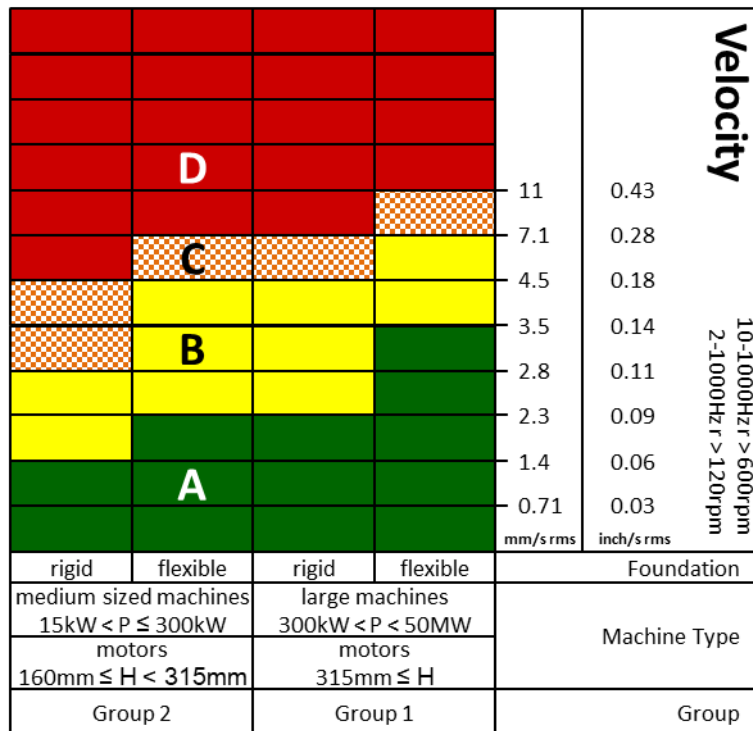
N = RPM/1000

R = Radius in cm

Unit Conversions

$D_{pk-pk} = \frac{19098 V_{pk}}{f_{cpm}} \quad V_{pk} = \frac{5217 A_{rms}}{f_{cpm}}$ $D_{pk-pk} = \frac{9.958 \times 10^7 A_{rms}}{f_{cpm}^2} \quad A_{rms} = \frac{f_{cpm} V_{pk}}{5217}$ $V_{pk} = \frac{f_{cpm} D_{pk-pk}}{19098} \quad A_{rms} = \frac{f_{cpm}^2 D_{pk-pk}}{9.958 \times 10^7}$	$D_{pk-pk} = \frac{27009 V_{rms}}{f_{cpm}} \quad V_{rms} = \frac{93712 A_{rms}}{f_{cpm}}$ $D_{pk-pk} = \frac{2.53 \times 10^9 A_{rms}}{f_{cpm}^2} \quad A_{rms} = \frac{f_{cpm} V_{rms}}{93712}$ $V_{rms} = \frac{f_{cpm} D_{pk-pk}}{27009} \quad A_{rms} = \frac{f_{cpm}^2 D_{pk-pk}}{2.53 \times 10^9}$
<p>D = Displacement: mils pk-pk V = Velocity: in/sec pk A = Acceleration: g rms F = Frequency: CPM</p>	<p>D = Displacement: micron pk-pk V = Velocity: mm/sec rms A = Acceleration: g rms F = Frequency: CPM 1g rms = 9.8m/sec²</p>

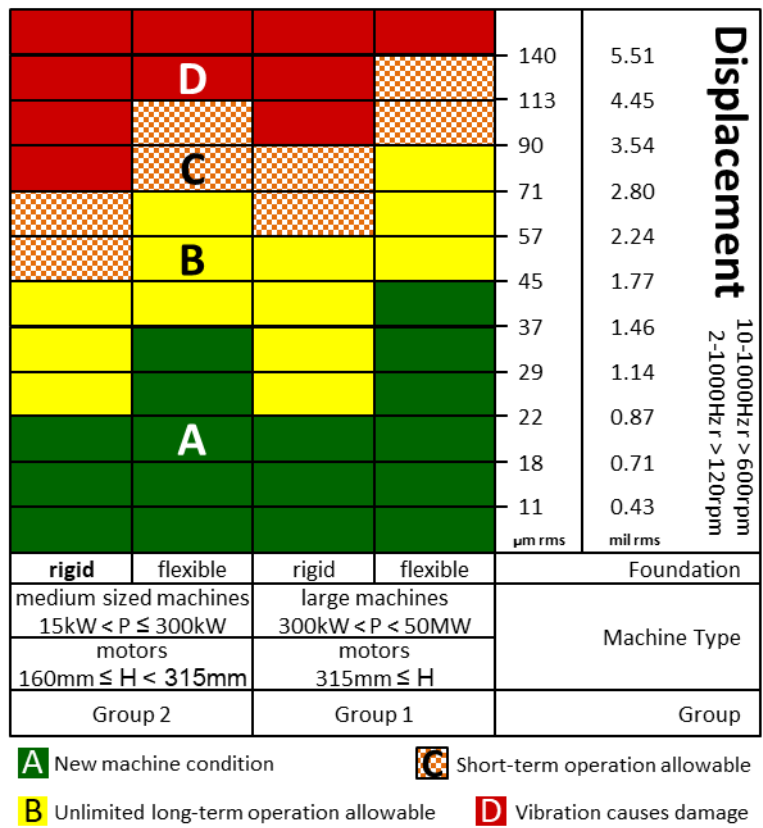
ISO 10816-3 Vibration Severity Chart



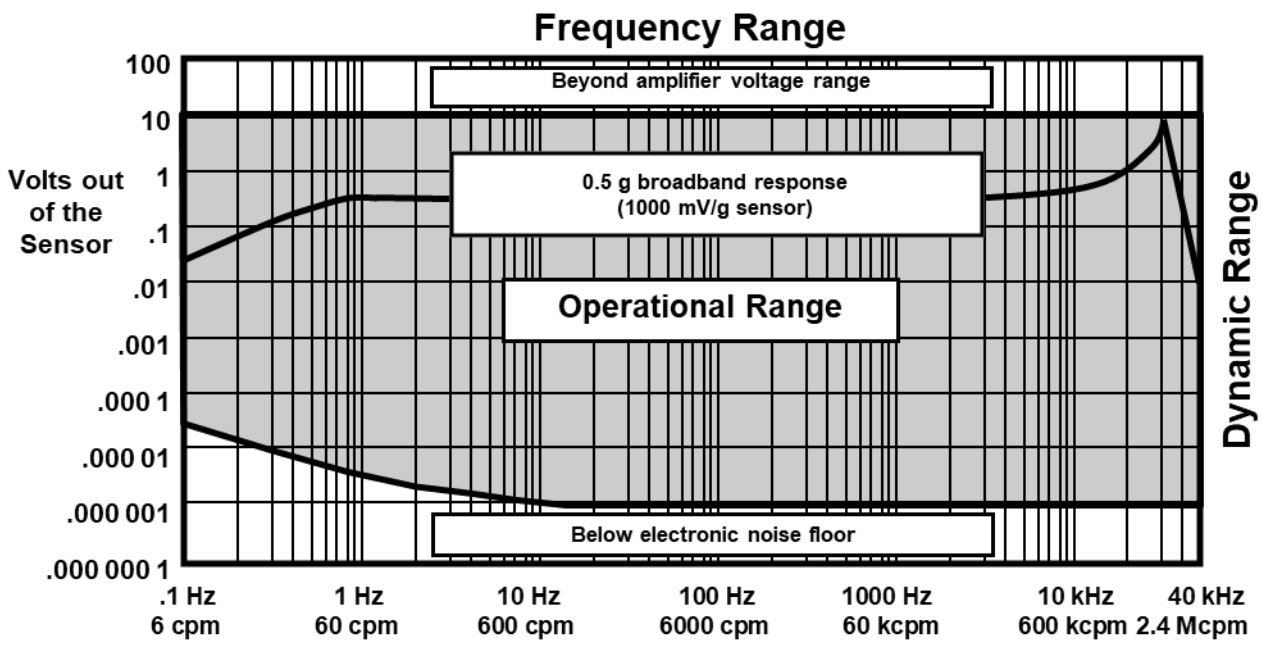
- A New machine condition
- B Unlimited long-term operation allowable
- C Short-term operation allowable
- D Vibration causes damage

If the lowest natural frequency of the combined machine and support system in the direction of measurement is higher than its main excitation frequency (this is in most cases the rotational frequency) by at least 25 %, then the support system may be considered rigid in that direction. All other support systems may be considered flexible.

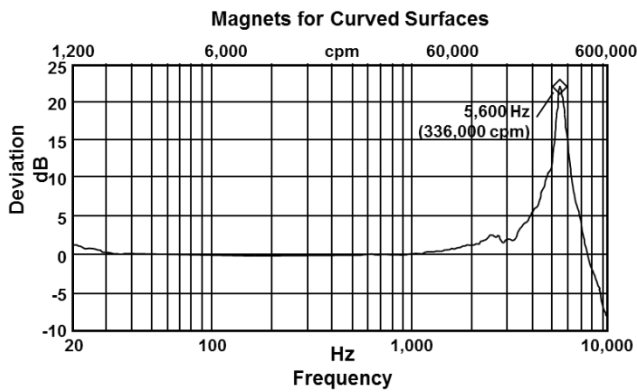
ISO 10816-3 Vibration Severity Chart



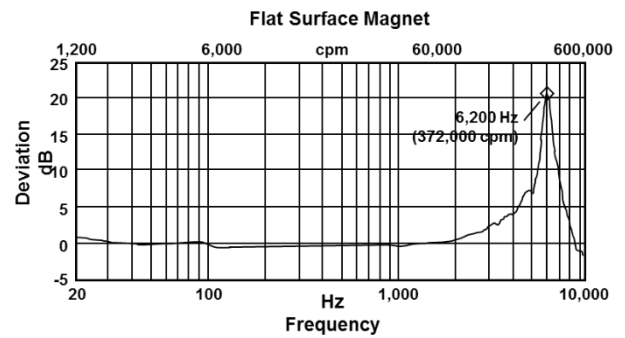
Transducer Operating Regions



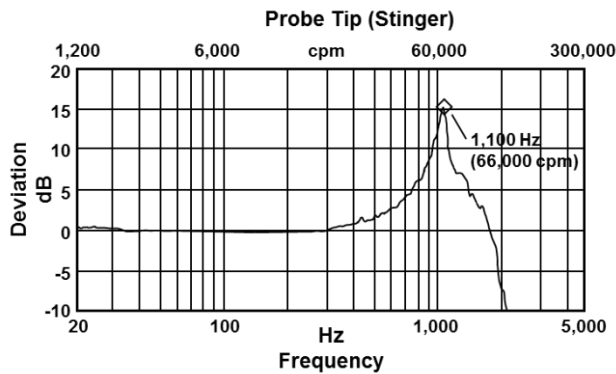
Transducer Frequency Response - Magnets on Curved Surfaces



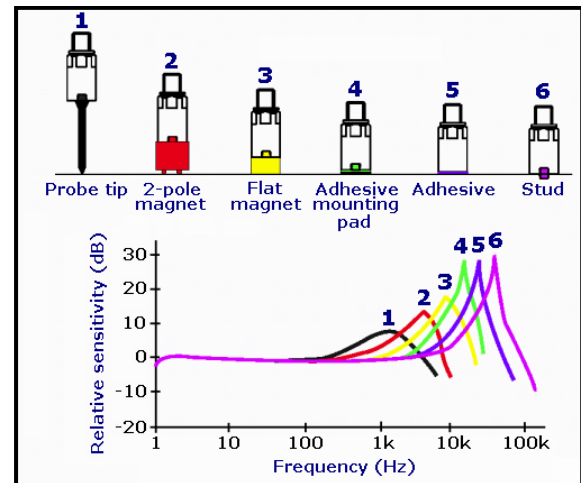
Transducer Frequency Response - Magnets on Flat Surfaces



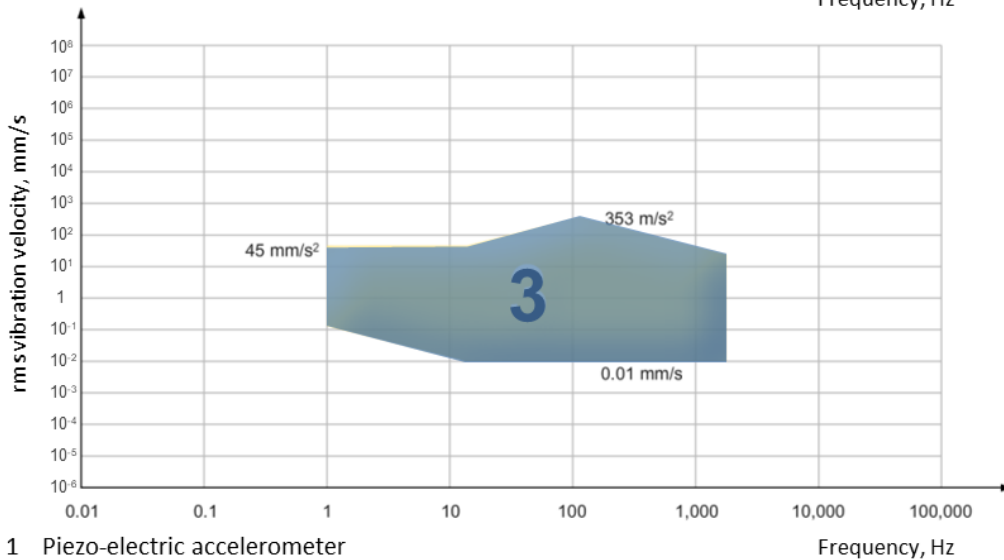
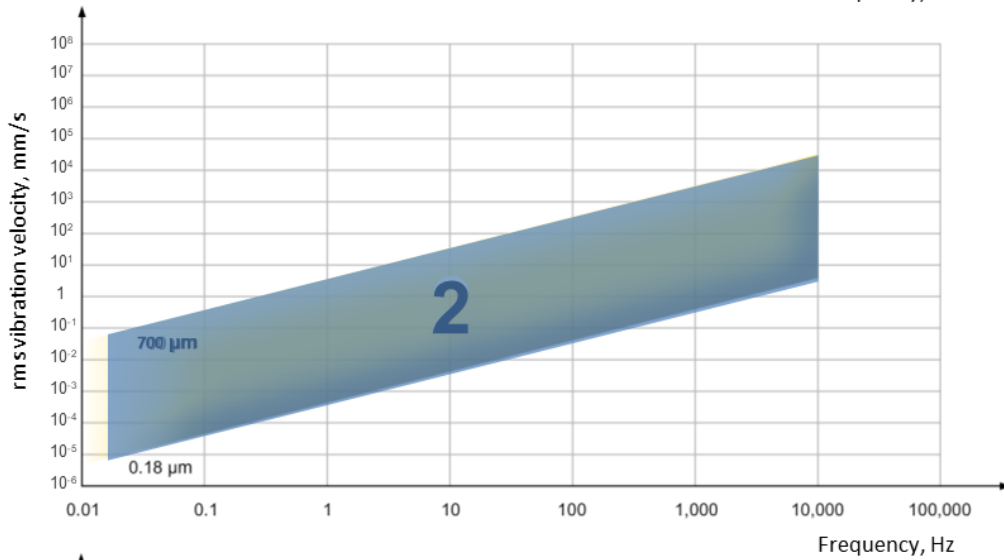
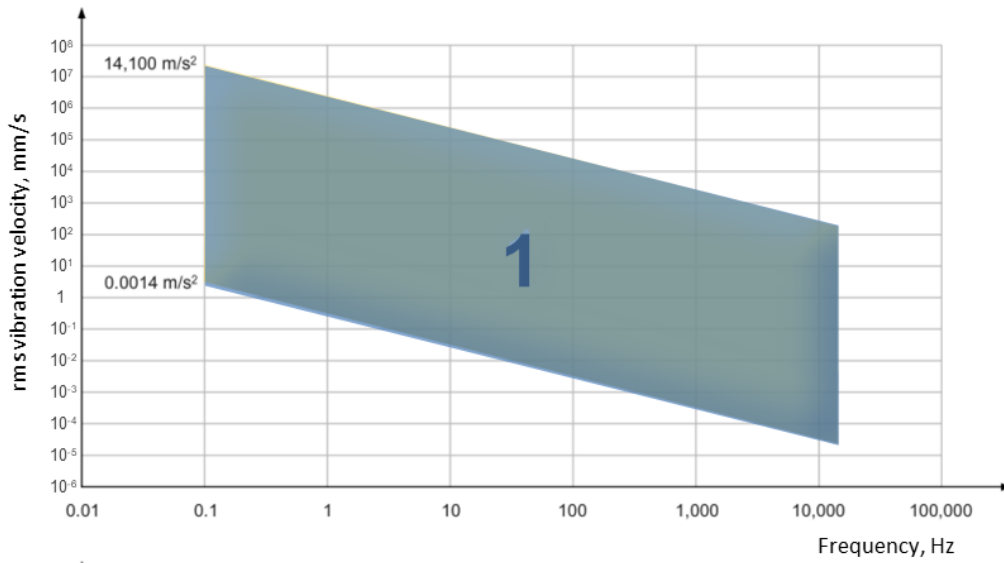
Transducer Frequency Response - Stingers or Hand Held Probes



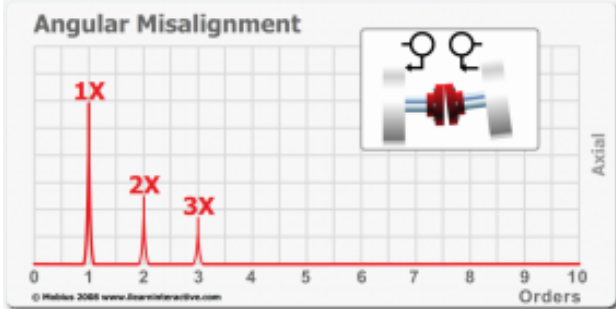
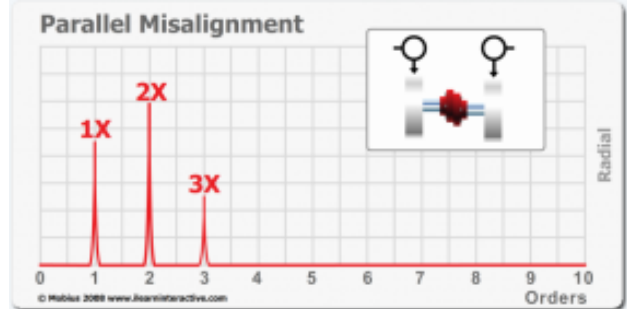
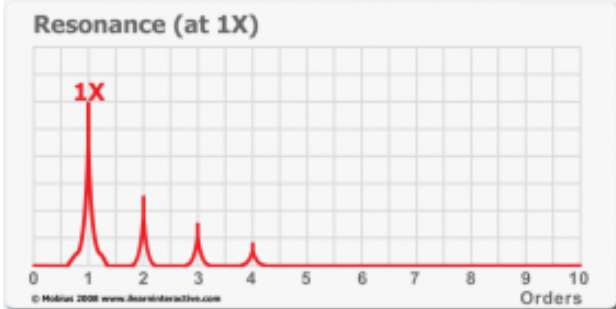
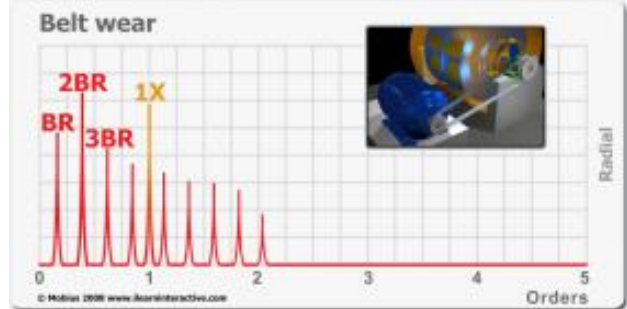
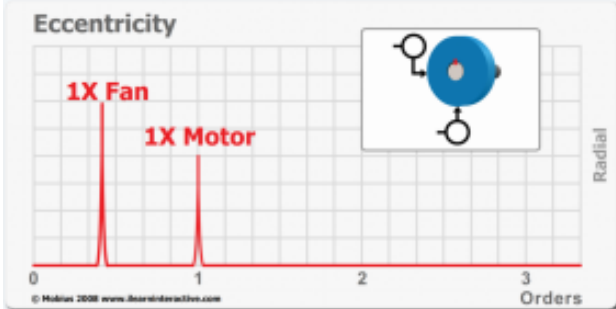
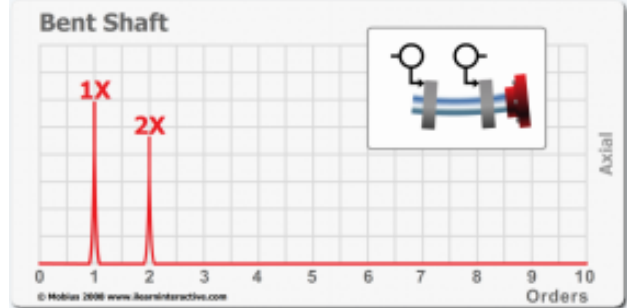
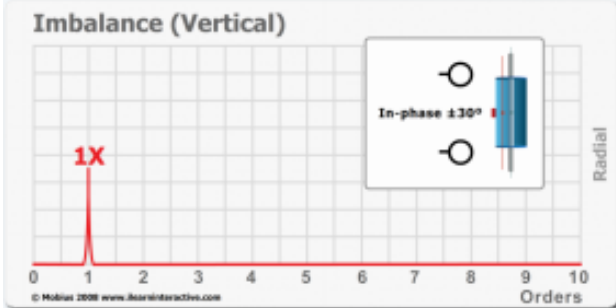
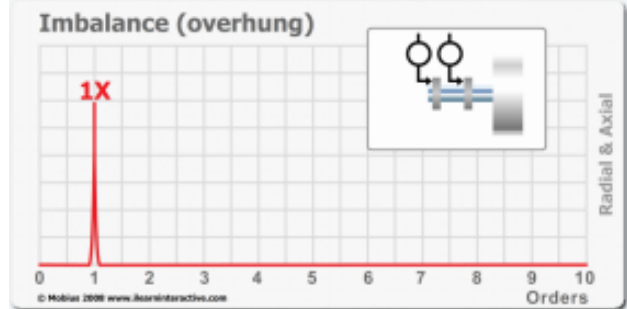
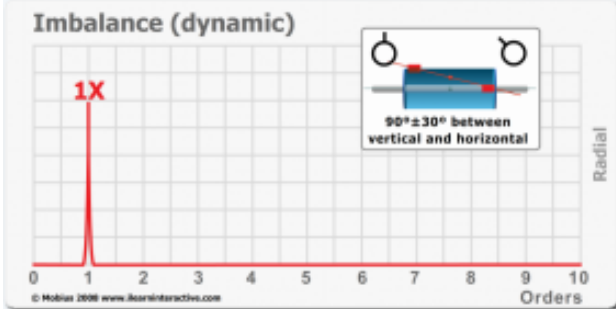
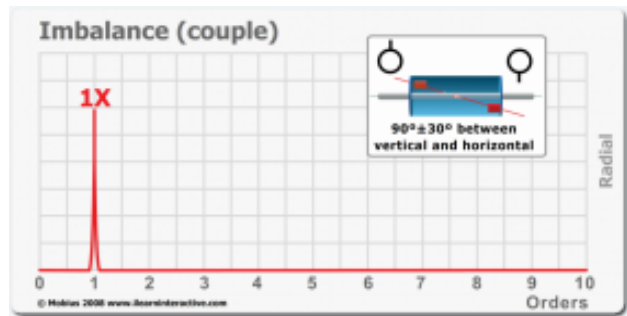
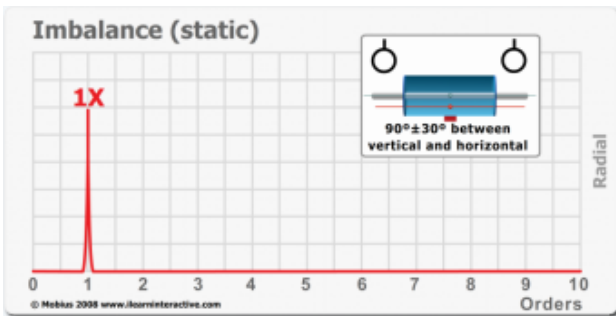
Transducer Frequency Response Curves

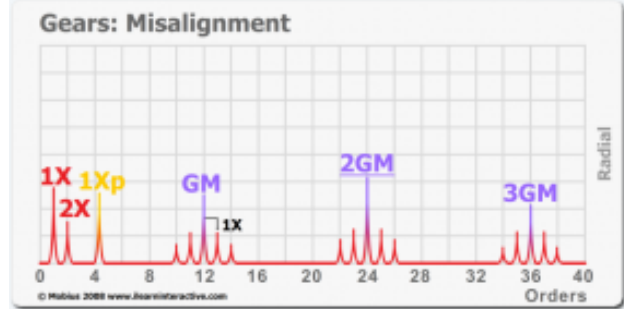
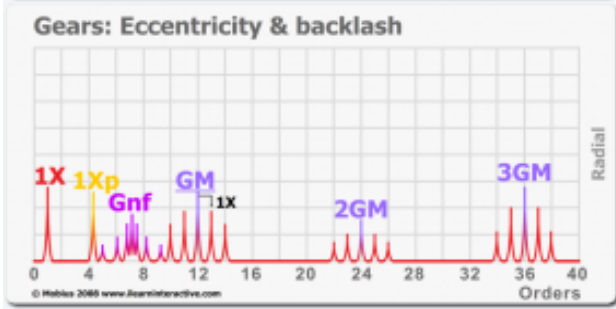
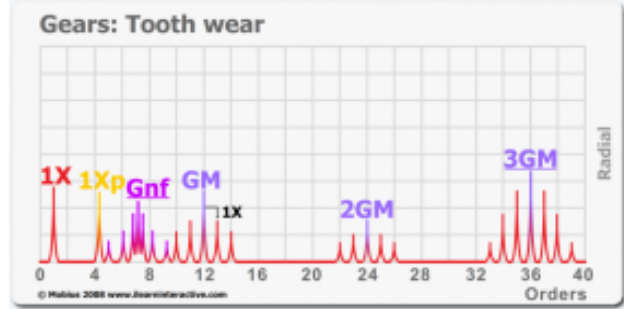
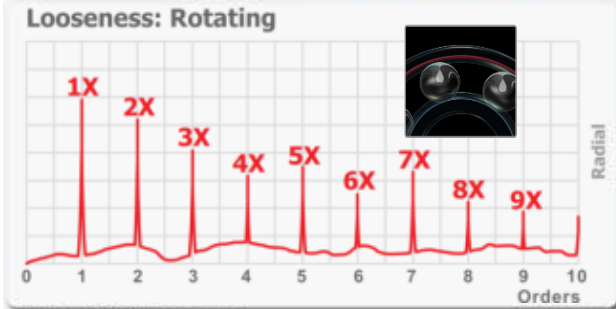
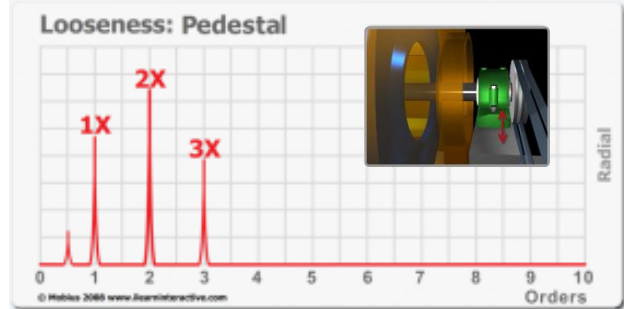
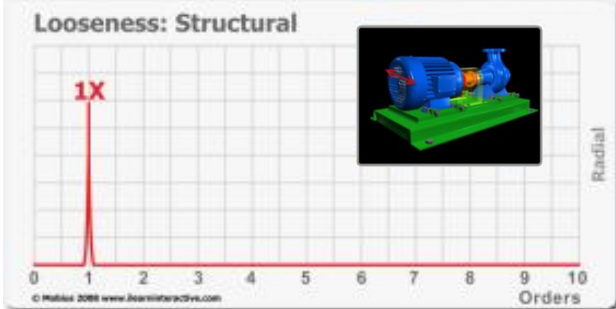
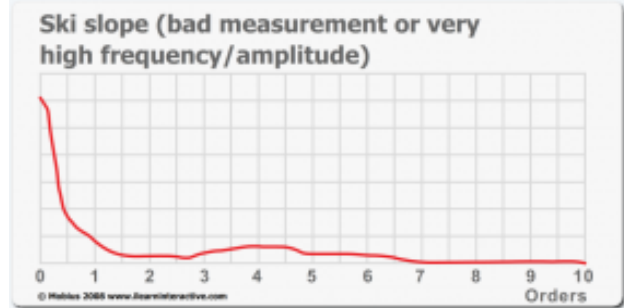
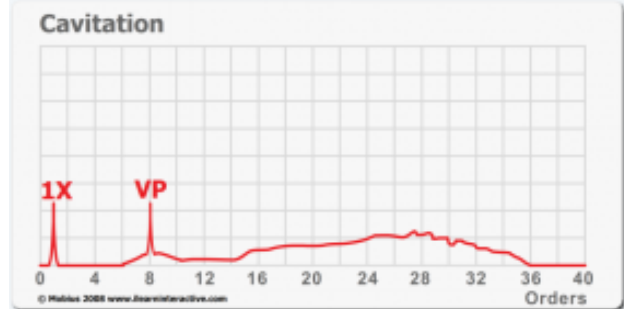
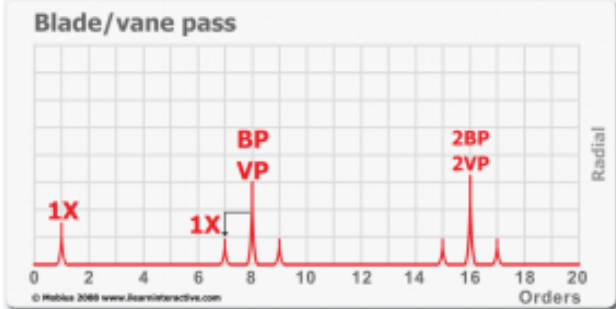
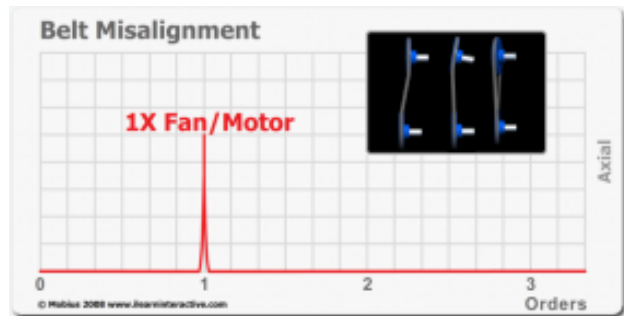
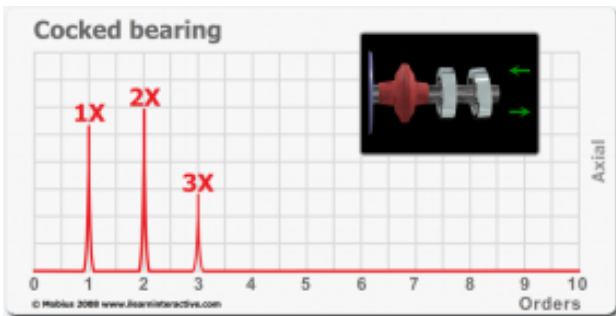


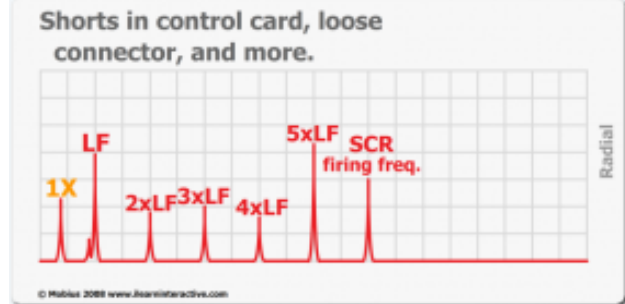
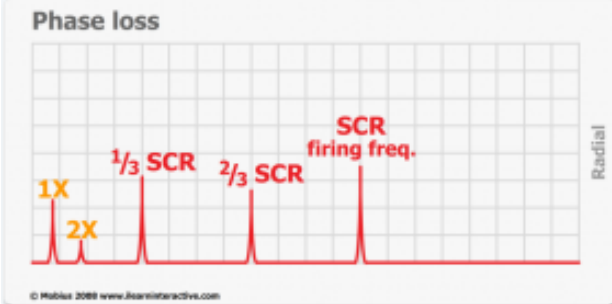
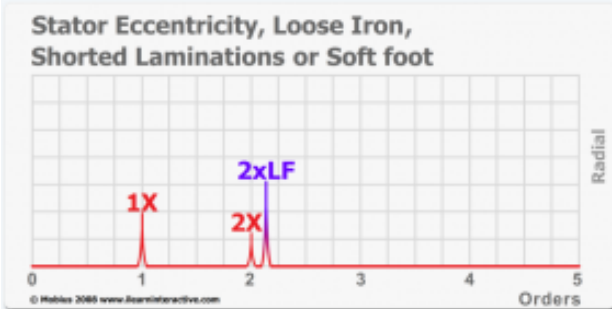
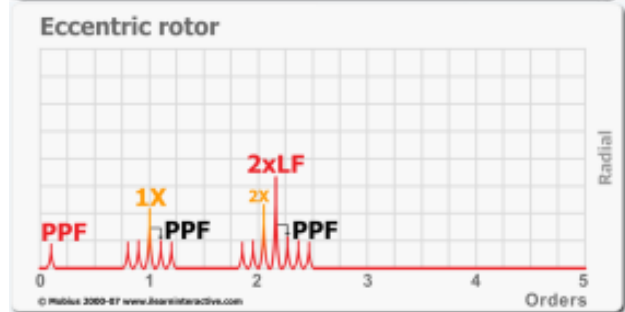
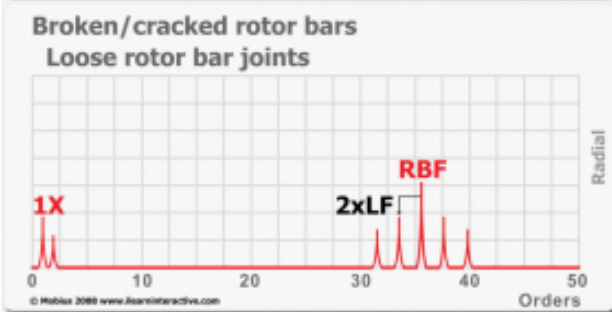
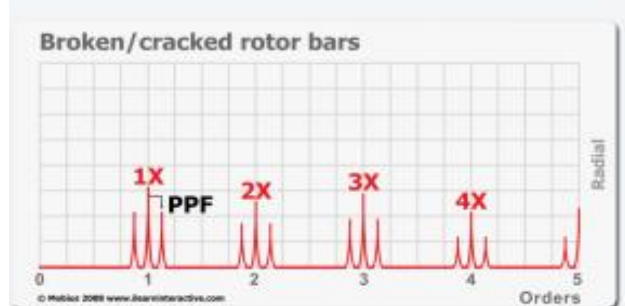
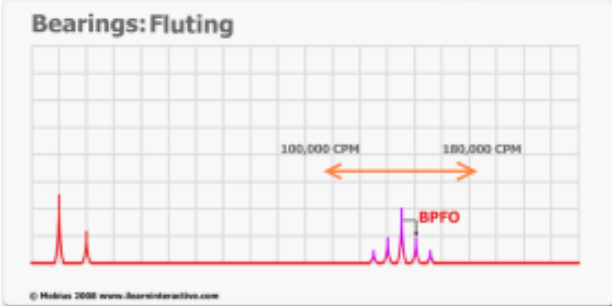
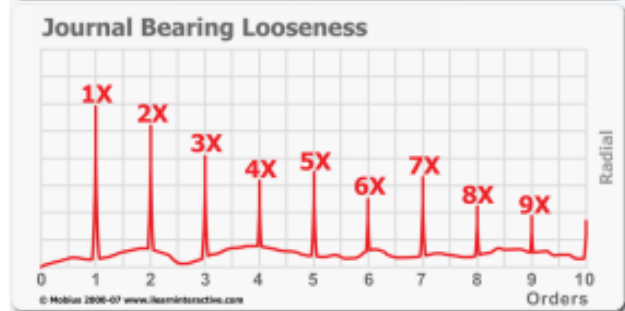
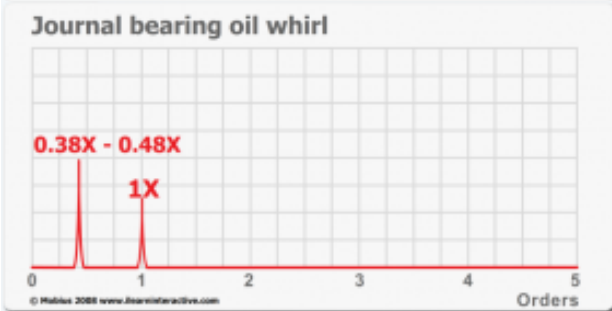
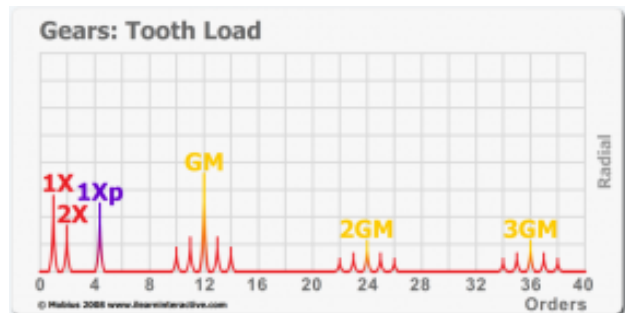
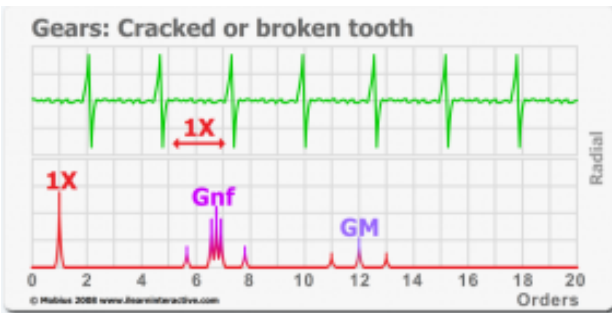
Transducer Effectiveness Regions

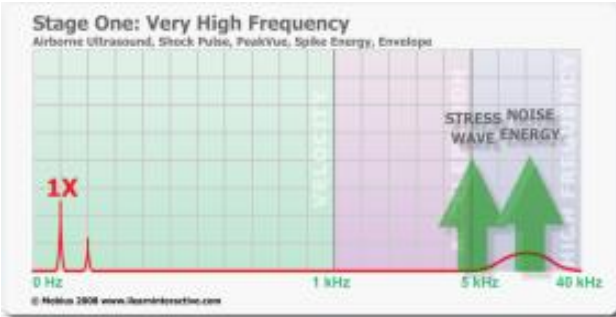
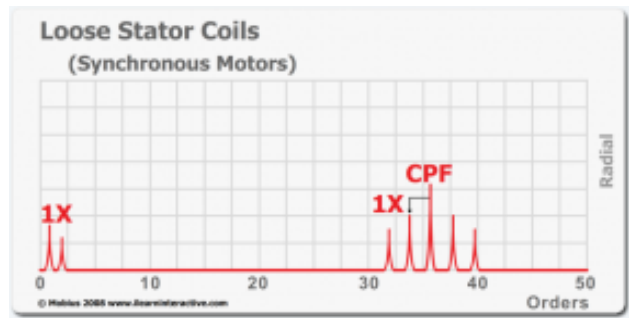
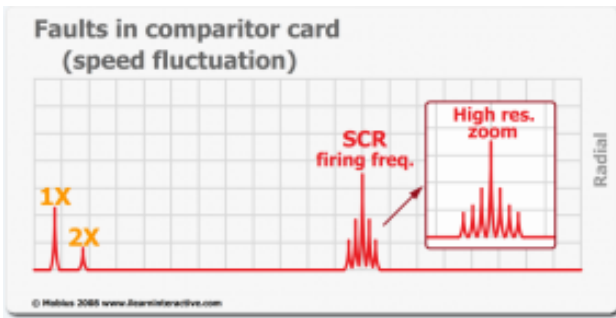


- 1 Piezo-electric accelerometer
- 2 Eddy-current proximity probe
- 3 Electro-mechanical velocity transducer

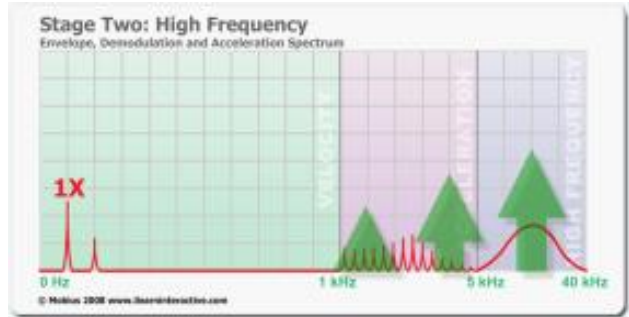




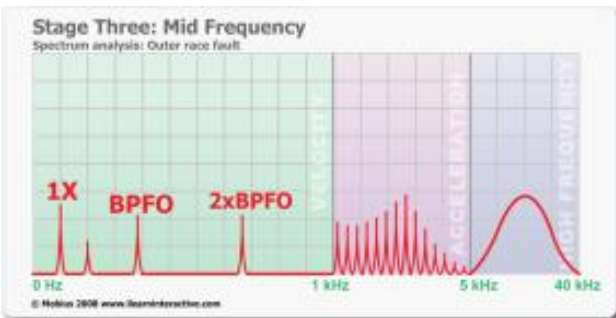




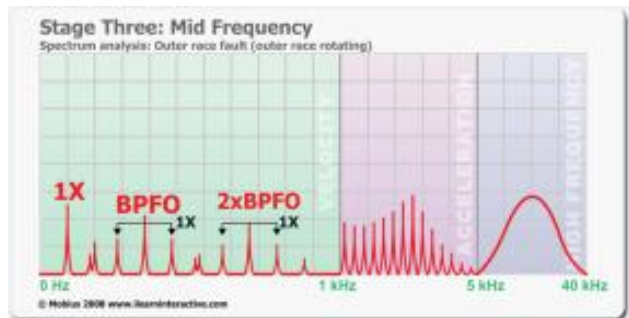
Stage One: Airborne Ultrasound, Shock Pulse, PeakVue, Spike Energy, Envelope



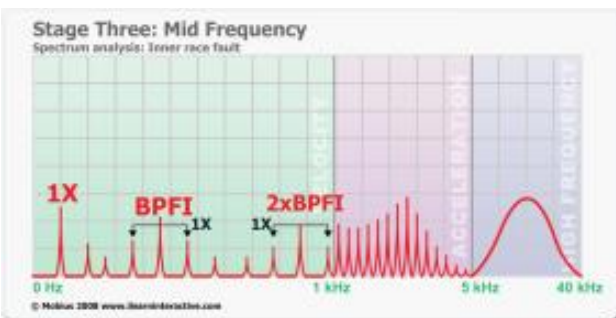
Stage Two: Envelope, Demodulation and Acceleration Spectrum



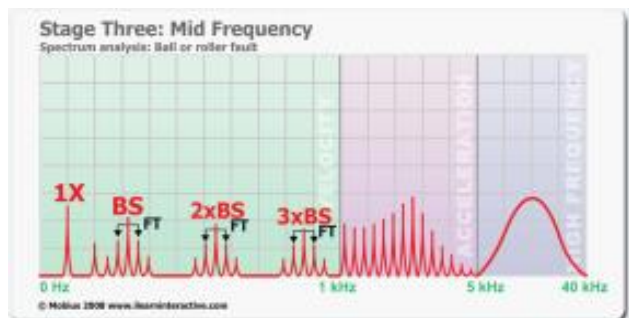
Stage Three: Outer race fault (inner race rotating)



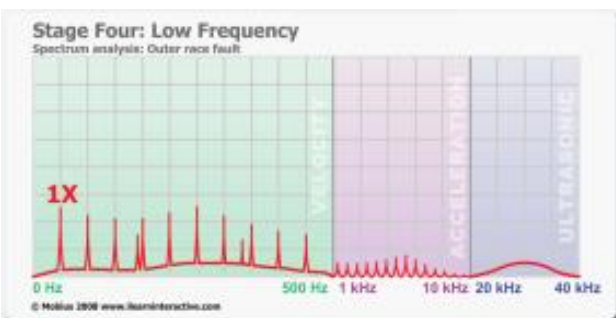
Stage Three: Outer race fault (outer race rotating)



Stage Three: Inner race fault (inner race rotating)



Stage Three: Ball or roller fault (inner race rotating)



Stage Four

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