

#### **BAHAN REFERENSI UJIAN SERTIFIKASI**

$$T = T_s \times N = \frac{N}{F_s} = \frac{N}{2.56 \times F_{max}} = \frac{lines}{F_{max}}$$

T = Waktu yang dibutuhkan untuk pengambilan data waveform

Ts = Waktu antara tiap samplel data

Fs = Laju sample data = Jumlah data per detik

N = Jumlah data sample (1024, 2048, 4096, etc.)

$$Resolusi = \frac{F_{max}}{lines}$$

## Bandwidth = Resolusi × Window factor

Window factor = 1 (no window/uniform/rectangular) or 1.5 (Hanning window)

Frekuensi pemisah (*separating freq.*)  $\ge$  2 x Bandwidth  $\ge$  2 x Resolusi \* Window Factor

Jumlah lines yang diperlukan di spectrum  $\ge 2 \times$  Window factor x Fmax / Frekuensi pemisah

Keakuratan frekuensi (di Peak) =  $\pm \frac{1}{2}$  x Resolution

Bilangan Prima: 1, 2, 3, 5, 7, 11, 13, 17, 19...

1 inch = 25.4 mm1 mm = 0.039 inches

#### Perhitungan Berat Trial:

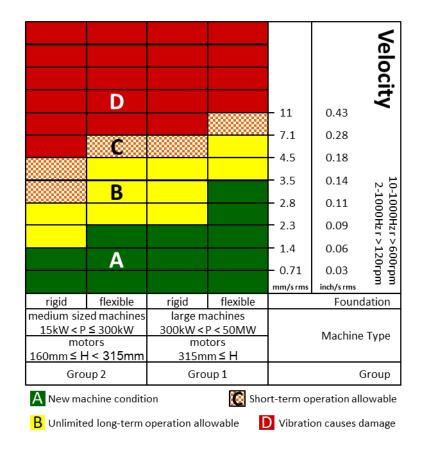
$$W = \frac{F}{K \times R \times N^2}$$

F = 10% dari berat rotor dibagi dengan jumlah bearing (kg) K = 0.011 N = RPM/1000R = Radius dalam cm

## Konversi satuan

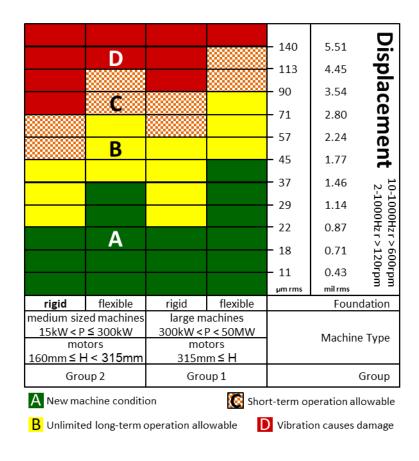
$D_{pk-pk} = \frac{19098 V_{pk}}{f_{cpm}}$ $V_{pk} = \frac{5217 A_{rms}}{f_{cpm}}$	$D_{pk-pk} = \frac{27009 V_{rms}}{f_{cpm}}$ $V_{rms} = \frac{93712 A_{rms}}{f_{cpm}}$
$D_{pk-pk} = \frac{9.958 \times 10^7 A_{rms}}{f_{cpm}^2} A_{rms} = \frac{f_{cpm}V_{pk}}{5217}$	$D_{pk-pk} = \frac{2.53 \times 10^9 A_{rms}}{f_{cpm}^2}  A_{rms} = \frac{f_{cpm} V_{rms}}{93712}$
$V_{pk} = \frac{f_{cpm}D_{pk-pk}}{19098} \qquad A_{rms} = \frac{f_{cpm}^2D_{pk-pk}}{9.958x10^7}$	$V_{rms} = \frac{f_{cpm}D_{pk-pk}}{27009} \qquad A_{rms} = \frac{f_{cpm}^2D_{pk-pk}}{2.53x10^9}$
<ul> <li>D = Displacement: mils pk-pk</li> <li>V = Velocity: in/sec pk</li> <li>A = Acceleration: g rms</li> <li>F = Frequency: CPM</li> </ul>	D = Displacement: micron pk-pk V = Velocity: mm/sec rms A = Acceleration: g rms F = Frequency: CPM 1g rms = 9.8m/sec <sup>2</sup>

# **ISO 10816-3 Vibration Severity Chart**

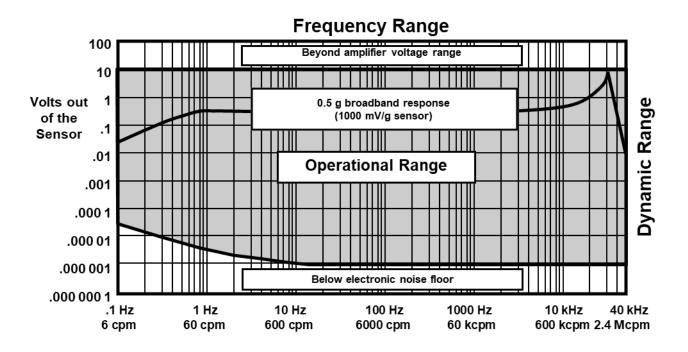


If the lowest natural frequency of the combined machine and support system in the direction of measurement is higher than its main excitation frequency (this is in most cases the rotational frequency) by at least 25 %, then the support system may be considered rigid in that direction. All other support systems may be considered flexible.

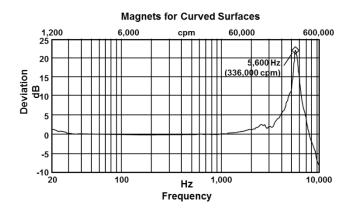
### **ISO 10816-3 Vibration Severity Chart**



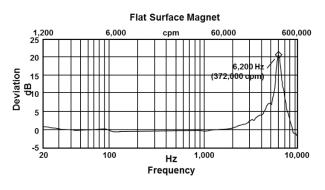
## **Transducer Operating Regions**



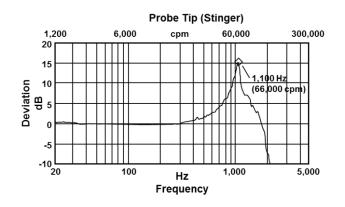
## Transducer Frequency Response - Magnets on Curved Surfaces



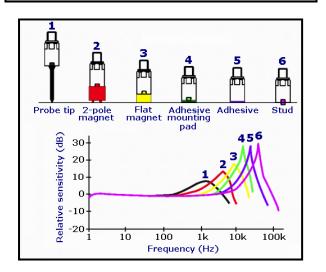
#### **Transducer Frequency Response -** Magnets on Flat Surfaces



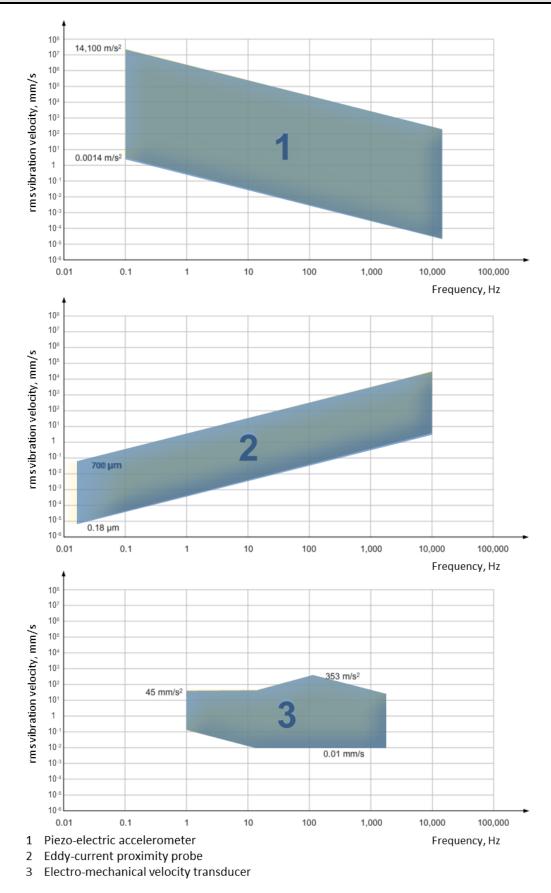
### **Transducer Frequency Response -** Stingers or Hand Held Probes

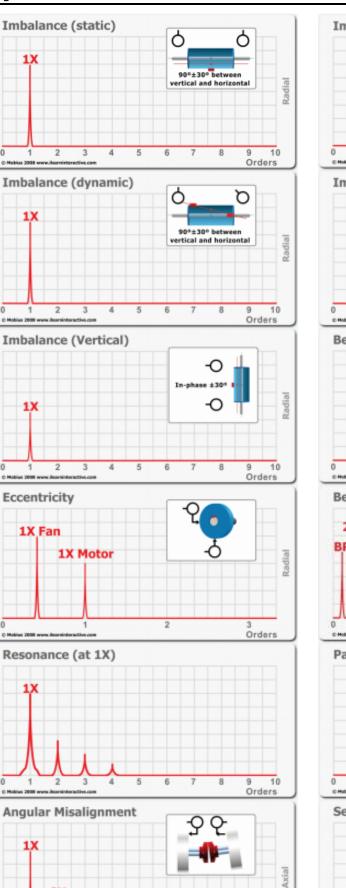


#### Transducer Frequency Response Curves

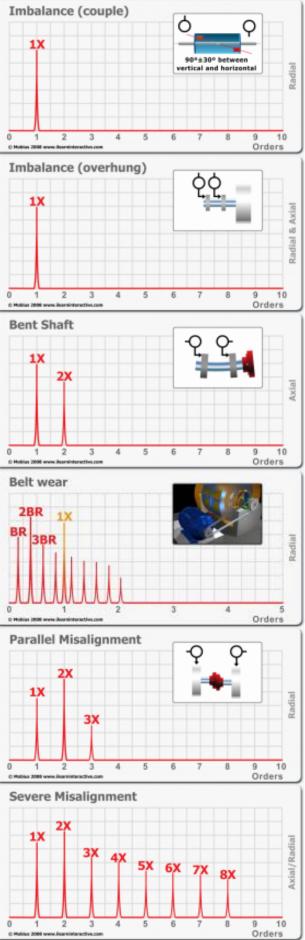


# **Transducer Effectiveness Regions**





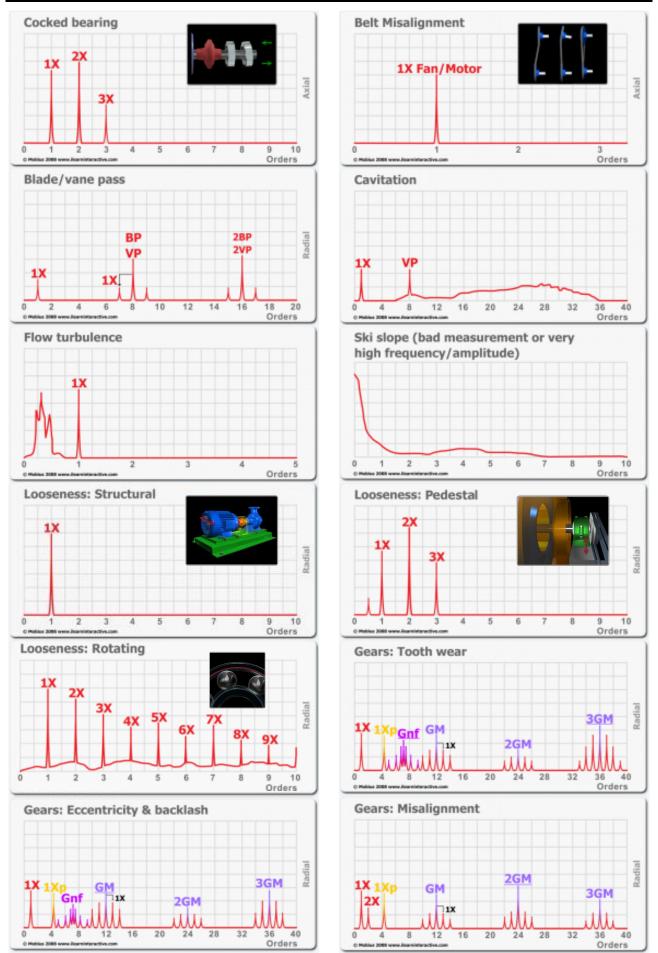
3X



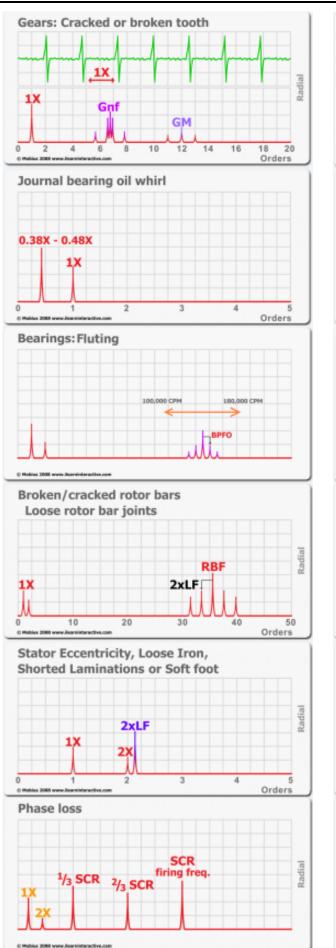
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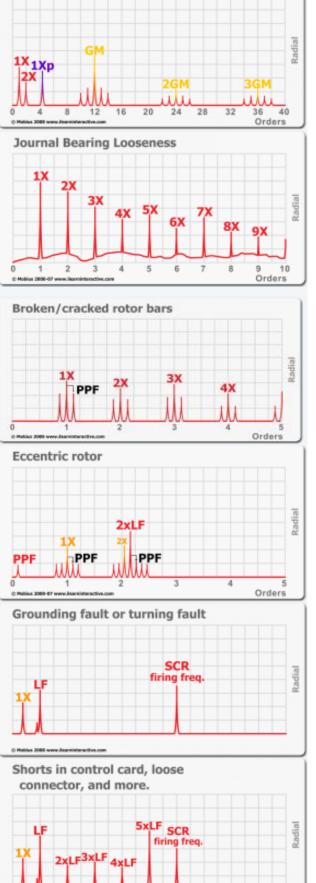
10

9 10 Orders



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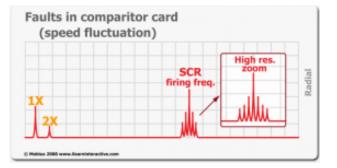
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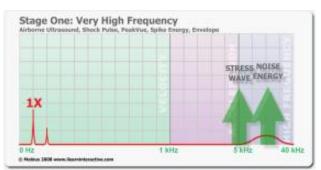
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**Certification Exam - Reference** 

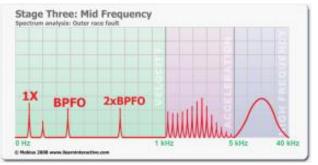
Gears: Tooth Load

#### **Certification Exam - Reference**

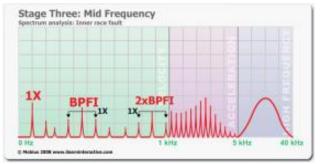




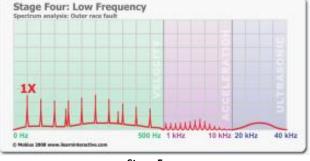
Stage One: Airborne Ultrasound, Shock Pulse, PeakVue, Spike Energy, Envelope



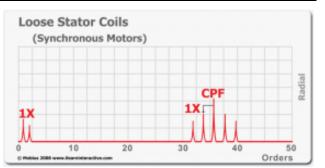
Stage Three: Outer race fault (inner race rotating)

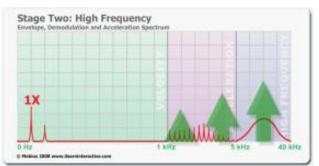


Stage Three: Inner race fault (inner race rotating)

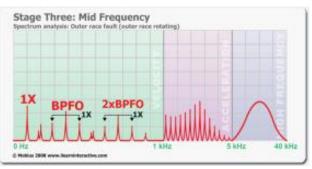


Stage Four

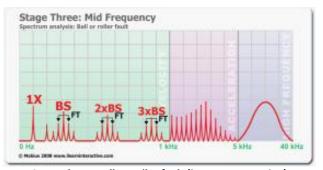




Stage Two: Envelope, Demodulation and Acceleration Spectrum



Stage Three: Outer race fault (outer race rotating)



Stage Three: Ball or roller fault (inner race rotating)

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